

New Aspects in Unsteady Cascade Aeroelasticity by Non-linear Singular Integral Equations using Steam Assisted Gravity Drainage (SAGD)

E.G. Ladopoulos
Interpaper Research Organization
8, Dimaki Str.
Athens, GR - 106 72, Greece
eladopoulos@interpaper.org

Abstract

A sophisticated non-linear model is further improved and investigated, for the determination of the velocity field around a cascade of airfoils of a turbomachine. Such problem is reduced to the solution of a non-linear multidimensional singular integral equation, when considering harmonic time dependence between the motions of adjacent blades of the turbine. Consequently, in order to increase the efficiency of the *Once Through Steam Generator* (OTSG) generating steam for the *Steam-Assisted Gravity Drainage* (SAGD) method of the petroleum sands, as well as to reduce the high Greenhaus Gas Emissions (GHG), a special turbomachine is studied. Especially, the velocity field around a cascade of airfoils is determined, while this problem is reduced to the solution of a non-linear multidimensional singular integral equation, when considering harmonic time dependence between the motions of adjacent blades of the turbine. Thus, a new non-linear model is investigated, by proposing a modern method.

Key Word and Phrases

Steam-Assisted Gravity Drainage (SAGD), Non-linear Singular Integral Equations, Unsteady Cascade Aeroelasticity, Once Through Steam Generator (OTSG), Petroleum Sands, Turbomachine, Airfoil, Greenhouse Gas Emissions (GHG).

1. Introduction – Improvements of Proposed Technology

Oil sands, or more technically, bituminous sands, are a type of unconventional petroleum deposit. Thus, the petroleum sand is either loose sand or partially consolidated sandstone containing a naturally occurring mixture of sand, clay, and water, saturated with a dense and extremely viscous form of petroleum technically referred to as bitumen. Natural bitumen deposits are reported in many countries, but in particular are found in extremely large quantities in Canada. The estimated worldwide deposits of petroleum are more than 2 trillion barrels and the estimates include deposits that haven't been discovered.

The crude bitumen contained in the Canadian oil sands is described by the National Energy Board of Canada as "a highly viscous mixture of hydrocarbons heavier than pentanes which, in its natural state, is not usually recoverable at a commercial rate through a well because it is too thick to flow." Crude bitumen is a thick, sticky form of crude petroleum, so heavy and viscous that it will not flow unless heated or diluted with lighter hydrocarbons such as light crude petroleum or natural-gas condensate. The World Energy Council (WEC) defines natural bitumen as "oil having a viscosity greater than 10,000 centipoise under reservoir conditions and an API gravity of less than 10° API".

Besides, according to the study ordered by the Government of Alberta and conducted by Jacobs Engineering Group, emissions from oil-sand crude are 12% higher than from conventional petroleum. The most common recovery process for producing petroleum from oil sands reserves is known as *Steam-Assisted Gravity Drainage* (SAGD). In this process, steam is generated within an *Once Through Steam Generator* (OTSG) at a central processing facility, transported to well pads, and injected below ground into a horizontal wellbore within the reservoir. The heat supplied by the steam warms the heavy oil in the reservoir, allowing it to flow via gravity drainage into a second underlying wellbore that captures the oil/water mixture and produces it to the surface. The resultant oil/water mixture is separated, and the process water is then recycled to produce boiler feedwater

for steam generation in the OTSG. Consequently, the Greenhouse Gas Emissions (GHG) are higher because continuous steam generation for SAGD operations consumes significant amounts of natural gas.

SAGD was developed in the 1980s by the Alberta Oil Sands Technology and Research Authority and fortuitously coincided with improvements in directional drilling technology that made it quick and inexpensive to do by the mid 1990s. In SAGD, two horizontal wells are drilled in the petroleum sands, one at the bottom of the formation and another about 5 meters above it. These wells are typically drilled in groups off central pads and can extend for miles in all directions. In each well pair, steam is injected into the upper well, the heat melts the bitumen, which allows it to flow into the lower well, where it is pumped to the surface.

Thus, SAGD has proved to be a major breakthrough in production technology since it allows very high oil production rates, and recovers up to 60% of the oil in place. Because of its economic feasibility and applicability to a vast area of oil sands, this method alone quadrupled North American petroleum reserves and allowed Canada to move to second place in world oil reserves after Saudi Arabia. Most major Canadian oil companies now have SAGD projects in production or under construction in Alberta's oil sands areas and in Wyoming.

The production of bitumen and synthetic crude oil emits more greenhouse gases than the production of conventional crude oil. A 2009 study by the consulting firm IHS CERA estimated that production from Canada's petroleum sands emits "about 5% to 15% more carbon dioxide, over the "well-to-wheels" (WTW) lifetime analysis of the fuel, than average crude oil."

A Stanford University study commissioned by the EU in 2011 found that oil sands crude was as much as 22% more carbon intensive than other fuels. Greenpeace says the oil sands industry has been identified as the largest contributor to greenhouse gas emissions growth in Canada, as it accounts for 40 million tons of CO₂ emissions per year.

Hence, in order to increase the efficiency of the OTSG generating steam for the SAGD method of the petroleum sands, as well as to reduce the high GHG, a special turbomachine is proposed. According to the proposed modern technology a two-dimensional unsteady cascade aeroelasticity is introduced for the investigation of flowfields of turbomachines generating steam. Especially, the velocity field around a cascade of airfoils is determined, while such a problem is reduced to the solution of a non-linear multidimensional singular integral equation, when considering harmonic time dependence between the motions of adjacent blades of the turbine. Hence, a general non-linear model is investigated, by proposing a modern method.

Consequently, by considering an airfoil or blade in an axial flow turbine or compressor which is running at some rotational speed, then because of the aerodynamic and structural performance, the blade has certain geometric properties defined by its length, root and tip fixation, possible mechanical attachment to other blades and by the chord, camber, thickness, stagger and profile shape which are functions of the radial coordinate. Beyond the above, the blade can be constructed in such a manner that the line of centroids and the line of shear centers are neither radial nor straight, but are defined by schedules of axial and tangential coordinates as functions of radius.

Over the last years several non-linear singular integral equations methods were used successfully by E.G. Ladopoulos [1] - [36] for the solution of many applied problems of petroleum engineering. Thus, by the current investigation, a non-linear model is proposed for the study of an improved steam generator used in order to increase the efficiency for petroleum sand productions, as well as to reduce the existing GHG.

2. Unsteady Cascade Aeroelasticity - Non-linear Singular Integral Equations

The aeroelastic problem of the axial flow steam turbine will be used for a modern steam generator for the SAGD method of petroleum sands. In the turbomachine the angle of attack of each rotor airfoil at each radius r is compounded of the tangential velocities of the airfoil section due to rotor rotation and the through flow velocity as modified in direction by the upstream stator row. (Fig.1)

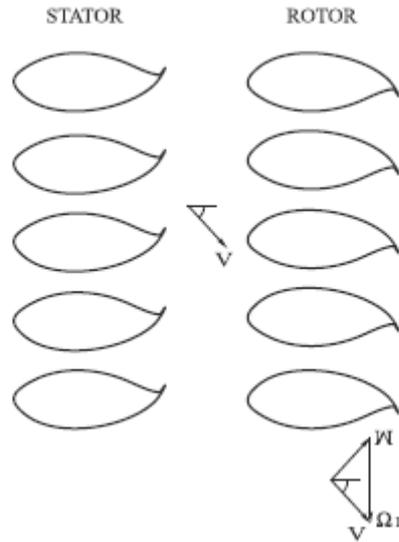


Fig. 1 Velocity triangle in an axial turbine

By considering a cascade of airfoils, then the fact that the flexible blades may be vibrating means that the relative pitch and stagger may be functions of time and also position in the cascade. In addition, a fundamental complication which further occurs is the necessity for treating the wakes of shed-vorticity from all the blades in the cascade and the flow under study is incompressible. (Fig. 2)

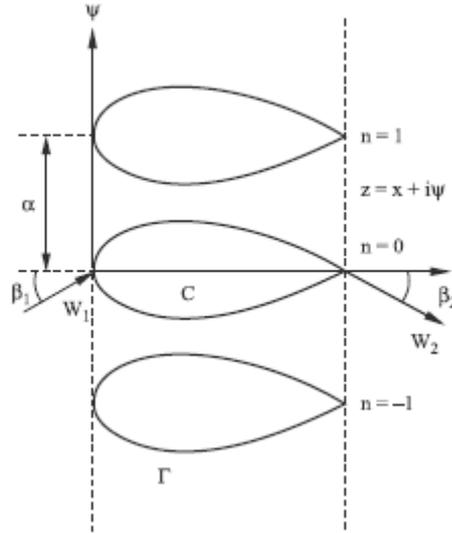


Fig. 2 A lattice of airfoils in an axial turbine

The velocities induced by an infinite column of vortices of equal strength Λ are given by the relation: [37]

$$\delta[u(z) - iv(z)] = \frac{i\Lambda}{2\pi} \sum_{n=-\infty}^{\infty} \frac{1}{Z - \zeta_n} \quad (2.1)$$

where ζ_n denotes the location of the n-th airfoil:

$$\zeta_n = \xi + ina e^{-i\beta} + iY_n(\xi_n, t) + X_n(t) \quad (2.2)$$

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in which t is the time and $Y_n \ll a$, $X_n \ll c$, with a the pitch (see: Fig 2) and c the length of the wake (Fig, 2). Also, the point Z is on the zeroth blade:

$$Z = x + iY(x, t) + X(t) \quad (2.3)$$

and the points ζ_n in (2.2) are equal to :

$$\xi_n = \xi + na \sin \beta \quad (2.4)$$

Furthermore, the harmonic time dependence between the motions of adjacent blades is given by the relation:

$$Y_n(\xi_n, t) = e^{i\pi\omega\tau} Y(\xi, t) \quad (2.5)$$

with τ the time lag.

Consequently, the singular kernel in (2.1) takes the form:

$$\frac{1}{Z - \zeta_n} = \frac{1}{[x - \xi - inae^{-i\beta} + i[Y(x, t) - Y_n(\xi_n, t)] + X(t) - X_n(t)]} \quad (2.6)$$

and for infinite number of blades:

$$\sum_{n=-\infty}^{\infty} \frac{1}{Z - \zeta_n} = \frac{1}{x - \xi + i[Y(x, t) - Y(\xi, t)]} + \sum_{n=-\infty}^{\infty} \frac{1}{Z - \zeta_n} \quad (2.7)$$

For the thin-airfoil theory the first term in (2.7) is conventionally ignored, and therefore the remaining term can be written as follows:

$$\begin{aligned} \sum_{n=-\infty}^{\infty} \frac{1}{Z - \zeta_n} &\approx \sum_{n=-\infty}^{\infty} \frac{1}{x - \xi - inae^{-i\beta}} + i \sum_{n=-\infty}^{\infty} \frac{Y_n(\xi_n, t) - Y(x, t)}{(x - \xi - inae^{-i\beta})^2} + \\ &+ \sum_{n=-\infty}^{\infty} \frac{X_n(t) - X(t)}{(x - \xi - inae^{-i\beta})^2} + \dots \end{aligned} \quad (2.8)$$

By combining (2.1) and (2.8) and denoting by $\lambda_a(x)$ (a =cascade spacing) the vortex strength distribution, one obtains for the unsteady induced velocities:

$$\begin{aligned} \delta[u^*(x') - iv^*(x')] &\approx -\frac{\lambda_a(\xi')\delta\xi'}{2\pi c} R^2 \left[\sum_{n=-\infty}^{\infty} \frac{e^{in\omega\tau} Y(\xi', t) - Y(x', t)}{(S - in\pi)^2} + \right. \\ &\left. + \frac{1}{i} \sum_{n=-\infty}^{\infty} \frac{e^{in\omega\tau} X(t) - X(t)}{(S - in\pi)^2} \right] \end{aligned} \quad (2.9)$$

where the variables R and S are equal to:

$$R = \pi e^{i\beta} (c/a) \quad (2.10a)$$

$$S = R(x' - \xi') \quad (2.10b)$$

and u^* , v^* are the time dependent parts of u and v .

Moreover, let us replace by Q :

$$Q = 1 - \frac{\omega\tau}{\pi} \quad (2.11)$$

and let the special case when the blades move perpendicular to their chordlines with the same amplitude along the chord:

$$Y = -c_1^* e^{i\omega t} = -c_2 \quad (2.12)$$

Additionally, by integrating over the chord in (2.9) we obtain the following non-linear multidimensional singular integral equation:

$$u^*(x') - iv^*(x') = \frac{R^2}{2\pi c} \int_0^1 \lambda_a(\xi') \sum_{n=-\infty}^{\infty} \frac{e^{in\omega\tau} c_2(t) - c_2(t)}{(S - in\pi)^2} \quad (2.13)$$

which further takes the form:

$$u^*(x') = -\frac{c_2}{2\pi c} \int_0^1 \lambda_a(\xi') [F - iI] d\xi' \quad (2.14a)$$

and:

$$v^*(x') = \frac{c_2}{2\pi c} \int_0^1 \lambda_a(\xi') [G + iH] d\xi' \quad (2.14b)$$

where:

$$F + iG = R^2 \frac{Q \sinh S \sinh QS - \cosh S \cosh QS + 1}{\sinh^2 S} \quad (2.15a)$$

and:

$$H + iI = R^2 \frac{Q \sinh S \cosh QS - \cosh S \sinh QS}{\sinh^2 S} \quad (2.15b)$$

3. Conclusions

By the present investigation a general non-linear model was proposed, for the determination of the velocity field around a cascade of airfoils of a turbomachine. Such problem was reduced to the solution of a non-linear multidimensional singular integral equation, when considering harmonic time dependence between the motions of adjacent blades of the turbine. Consequently, in order to increase the efficiency of the OTSG generating steam for the SAGD method of the petroleum sands, as well as to reduce the high GHG, a special turbomachine is proposed

Additionally, a mathematical model has been presented as an attempt to determine the properties of the OTSG turbine. The above mentioned problem was reduced to the solution of a non-linear singular integral equation, which was numerically evaluated by using a special software as already proposed by Interpaper Research Organization. So, there is a technical maturity of the proposed technique, as it based on ready software.

Such non-linear singular integral equation method will be of increasing interest in future, as these methods are very important for the solution of generalized solid mechanics and fluid mechanics problems. Modern problems of fluid and solid mechanics are much more simplified when solved by general non-linear singular integral equation methods.

Furthermore, the field of aeroelasticity in turbomachines, continues to be under active investigation, driven by the needs of steam turbine designers. Especially, the design of the new

generation turbomachines will be only possible by high sophisticated non-linear computational methods. Consequently, the non-linear singular integral equations method which was successfully used over the last years for the solution of problems of aerodynamics, fluid mechanics, hydraulics, structural analysis and fracture mechanics, etc. will be further used for the design of the next generation turbomachines, used for the SAGD method of the oil sands.

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